

## ME325 EXAM I (Sample)

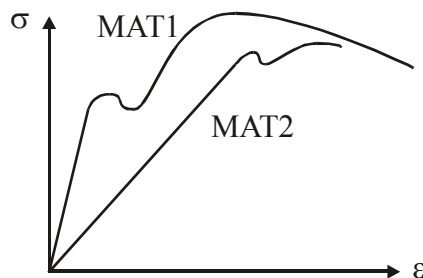
NAME: \_\_\_\_\_

NOTE:

- CLOSED BOOK, CLOSED NOTES. ONLY A SINGLE 8.5x11" FORMULA SHEET IS ALLOWED. ADDITIONAL INFORMATION IS AVAILABLE ON THE LAST PAGE OF THIS EXAM.
- DO YOUR WORK ON THE EXAM ONLY (NO SCRATCH PAPER ALLOWED).
- READ THE QUESTION AND ALL ANSWERS CAREFULLY AND SELECT THE **BEST** ANSWER.
- ALL QUESTIONS ARE WEIGHTED EQUALLY.

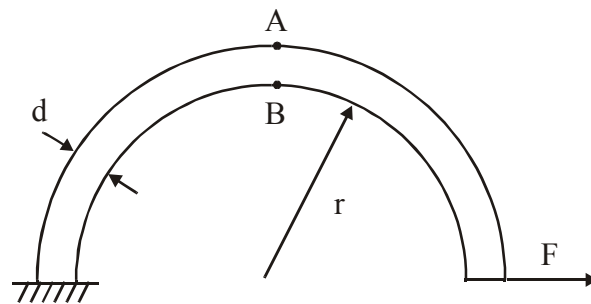
- (1) If a motor is turning a shaft with a torque of  $T$  ft-lb at a rate of  $n$  rpm, how much horsepower is being delivered?
- (a)  $\pi Tn / 550$
  - (b)  $550\pi Tn$
  - (c)  $\pi Tn / 16500$
  - (d)  $16500\pi Tn$
- (2) If an axial force is specified in units of kN and the cross section area is specified in  $\text{mm}^2$ , what is the appropriate unit for axial stress?
- (a) mPa
  - (b) Pa
  - (c) kPa
  - (d) MPa
  - (e) GPa

Questions 3 through 4 deal with the figure below:



- (3) Which material has a higher yield strength?  
 (a) MAT1  
 (b) MAT2
- (4) Which material has a higher ultimate strength?  
 (a) MAT1  
 (b) MAT2

Questions 5 through 8 deal with the curved rod of diameter  $d$  and inner radius of curvature  $r$  shown below.

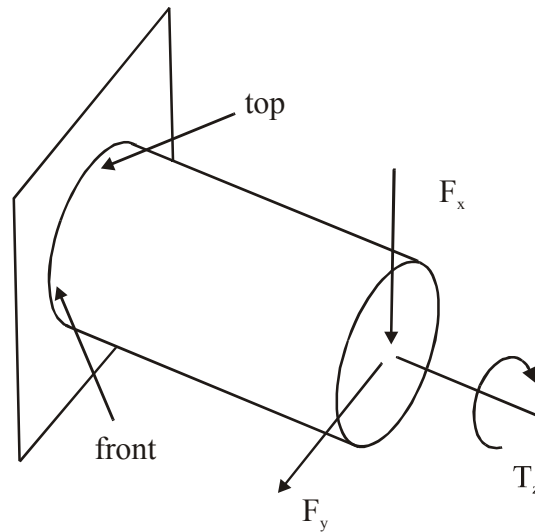


- (5) The bending moment  $M$  used to calculate the bending stress at point A is  
 (a)  $dF$   
 (b)  $(d/2)F$   
 (c)  $rF$   
 (d)  $(r + d/2)F$   
 (e)  $(r + d)F$
- (6) The  $\bar{r}/\bar{c}$  ratio used to determine the K factors is  
 (a)  $r/d$   
 (b)  $r/(d/2)$   
 (c)  $(r + d)/(d/2)$   
 (d)  $(r + d/2)/d$   
 (e)  $(r + d/2)/(d/2)$
- (7) The total normal stress at point B is calculated as  
 (a)  $|\sigma_{\text{bending}}| + |\sigma_{\text{axial}}|$   
 (b)  $|\sigma_{\text{bending}}| - |\sigma_{\text{axial}}|$   
 (c)  $|\sigma_{\text{axial}}| - |\sigma_{\text{bending}}|$   
 (d)  $-|\sigma_{\text{axial}}| - |\sigma_{\text{bending}}|$
- (8) How does the magnitude of the actual normal stress due to bending at point A compare with the magnitude that would be obtained using the straight beam equation?  
 (a) it is larger  
 (b) it is smaller  
 (c) it is the same

Questions 9 through 12 deal with the bar of length  $L$  and diameter  $d$  shown below. The left end of the bar is fixed to a wall. The free end is loaded with two forces and a torque as shown. The stress magnitudes at the wall (not including signs for direction) will be referred to below using the following labels:

- A: maximum bending stress magnitude due to  $F_x$
- B: maximum bending stress magnitude due to  $F_y$
- C: maximum torsion shear stress magnitude due to  $T_z$
- D: maximum transverse shear stress magnitude due to  $F_x$
- E: maximum transverse shear stress magnitude due to  $F_y$

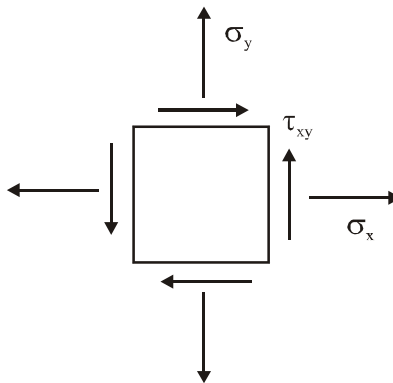
Note: the points labeled "top" and "front" are on the outer surface of the part on the vertical and horizontal planes of symmetry of the cross section.



- (9) What is the value of "A: maximum bending stress magnitude due to  $F_x$ "?
- (a)  $32F_xL / \pi d^3$
  - (b)  $64F_xL / \pi d^3$
  - (c)  $32F_xL / \pi d^4$
  - (d)  $16F_xL / \pi d^3$
  - (e)  $128F_xL / \pi d^4$

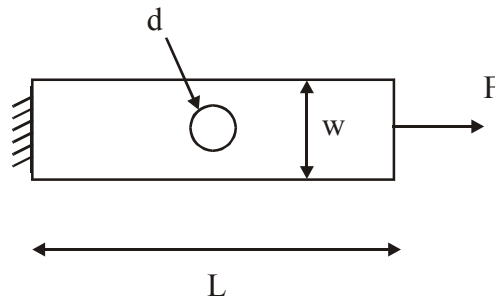
- (10) The total normal stress at the point labeled "front" is equal to
- A
  - B
  - A
  - B
  - A - B
- (11) The magnitude of the total shear stress at the point labeled "top" is equal to
- C
  - E + C
  - D + C
  - E - C
  - D - C
- (12) The transverse shear due to  $F_y$  is zero on the
- top surface
  - front surface

Questions 13 through 15 deal with the biaxial state of stress illustrated below where  $\sigma_z=0$ :



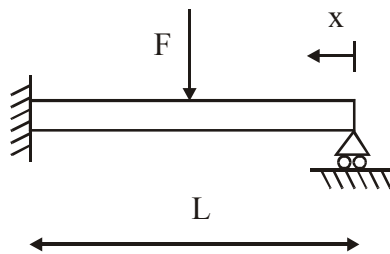
- (13) If  $\sigma_x = 100$ ,  $\sigma_y = -100$ , and  $\tau_{xy} = 0$ , the maximum shear stress in the material is
- 0
  - 50
  - 75
  - 100
  - 200
- (14) If  $\sigma_x = 100$ ,  $\sigma_y = 100$ , and  $\tau_{xy} = 50$ , the maximum shear stress in the material is
- 0
  - 50
  - 75
  - 100
  - 200

- (15) If  $\sigma_x = 100$ ,  $\sigma_y = -100$ , and  $\tau_{xy} = 50$ , which direction below corresponds to the smallest angle to the maximum principal stress?
- clockwise from  $\sigma_x$
  - counterclockwise from  $\sigma_x$
  - clockwise from  $\sigma_y$
  - counterclockwise from  $\sigma_y$
- (16) If the plate below has a thickness  $t$  and a stress concentration factor of 2, what is the correct expression for the maximum normal stress at the hole?



- $2F / t(w - d)$
- $F / 2t(w - d)$
- $2F / t(w - d/2)$
- $F / 2t(L - d)$
- $2F / t(L - d)$

Questions 17 through 19 deal with the beam shown below. Load  $F$  is applied at the center of the beam. Refer to the vertical reaction force at the right support as  $R_y$ . **NOTE:  $x$  is measured from the right side of the beam.** Neglect transverse shear effects.



- (17) What is the correct expression for the bending moment  $M(x)$  for  $0 < x < L/2$  (using the positive bending moment sign convention)?
- $xR_y$
  - $xR_y - FL/2$
  - $xR_y + FL/2$
  - $FL/2$
  - $-FL/2$

(18) Which equation below is valid and can be used to help solve for the reaction force  $R_y$ ?

(a)  $0 = \frac{1}{EI} \int_0^{L/2} M \frac{\partial M}{\partial F} dx$

(b)  $0 = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial F} dx$

(c)  $L = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial R_y} dx$

(d)  $F = \frac{1}{EI} \int_0^{L/2} M \frac{\partial M}{\partial R_y} dx$

(e)  $0 = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial R_y} dx$

(19) Which expression below can be used to find the deflection at the center of the beam?

(a)  $\frac{1}{EI} \int_0^L M \frac{\partial M}{\partial F} dx$

(b)  $\frac{1}{EI} \int_0^{L/2} M \frac{\partial M}{\partial F} dx$

(c)  $\frac{1}{EI} \int_0^{L/2} M \frac{\partial M}{\partial R_y} dx$

(d)  $\frac{1}{EI} \int_0^L M \frac{\partial M}{\partial R_y} dx$

(20) If  $M(x) = Px$ , what is  $\int_0^L M \frac{\partial M}{\partial P} dx$

(a)  $PL$

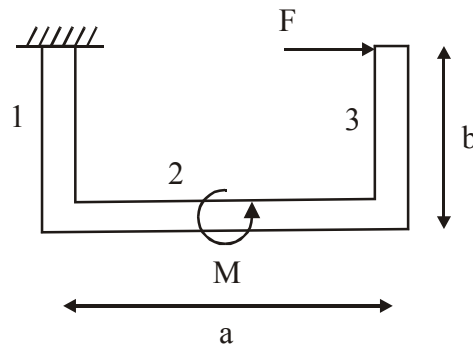
(b)  $PL^2$

(c)  $PL^2/2$

(d)  $PL^3$

(e)  $PL^3/3$

Questions 21 through 24 deal with applying Castigliano's Method to the frame below consisting of 3 segments. **Do not neglect any elastic energy terms and include all existing elastic terms (even if the partial derivative terms are zero).**



- (21) What is the reaction bending moment at the fixed end of section 1?
- $M + aF$
  - $M + bF$
  - $M - aF$
  - $M - bF$
  - $M$
- (22) In finding the horizontal deflection of the free end of section 3, what energy terms need to be included for section 1?
- none
  - transverse shear only
  - axial and bending only
  - transverse shear and bending only
  - axial, transverse shear, and bending
- (23) In finding the horizontal deflection of the free end of section 3, what energy terms need to be included for section 2?
- none
  - axial only
  - bending only
  - axial and bending only
  - axial, transverse shear, and bending
- (24) In finding the vertical deflection of the free end of section 3, what energy terms need to be included for section 3?
- none
  - transverse shear only
  - axial only
  - axial and transverse shear only
  - axial, transverse shear, and bending