

CE 716 Homework 2

1. Calculate the lift force in SI and English units on a 4 m-diameter semi-spherical tent under a 100 km/h wind. Compare with the lift force of a 4 m-long semi-cylindrical tent that has the same volume. (Hint: find the mass density of air and assume that there is no velocity at the base of the tent.)

For a semi-sphere:

$$F_L = \frac{27}{32} \rho v_\infty^2 \pi R^2$$

$$F_L = \frac{27}{32} \left( \frac{1.204 \text{ kg}}{\text{m}^3} \right) \left( 100 \frac{\text{km}}{\text{hr}} * \frac{1000 \text{ m}}{1 \text{ km}} * \frac{1 \text{ hr}}{60^2 \text{ s}} \right)^2 \pi * \left( \frac{4}{2} \text{ m} \right)^2 = \boxed{9,850 \text{ N}}$$

$$9850 \text{ N} * \left( 0.225 \frac{\text{lb}}{\text{N}} \right) = 2,240 \text{ lb}$$

For a cylinder, we can find the radius by using the volume of the half sphere

$$V = \frac{2}{3} \pi R^3 = \frac{2}{3} \pi \left( \frac{4}{2} \text{ m} \right)^3 = 16.76 \text{ m}^3$$

$$16.76 \text{ m}^3 = \frac{1}{2} \pi R^2 L$$

$$16.76 \text{ m}^3 = \frac{1}{2} \pi R^2 4 \text{ m}$$

$$R = \sqrt{ \frac{16.76 \text{ m}^3 * 2}{\pi * 4 \text{ m}} } = 1.63 \text{ m}$$

Now we can plug into the lift equation E-4.2.2 from the book:

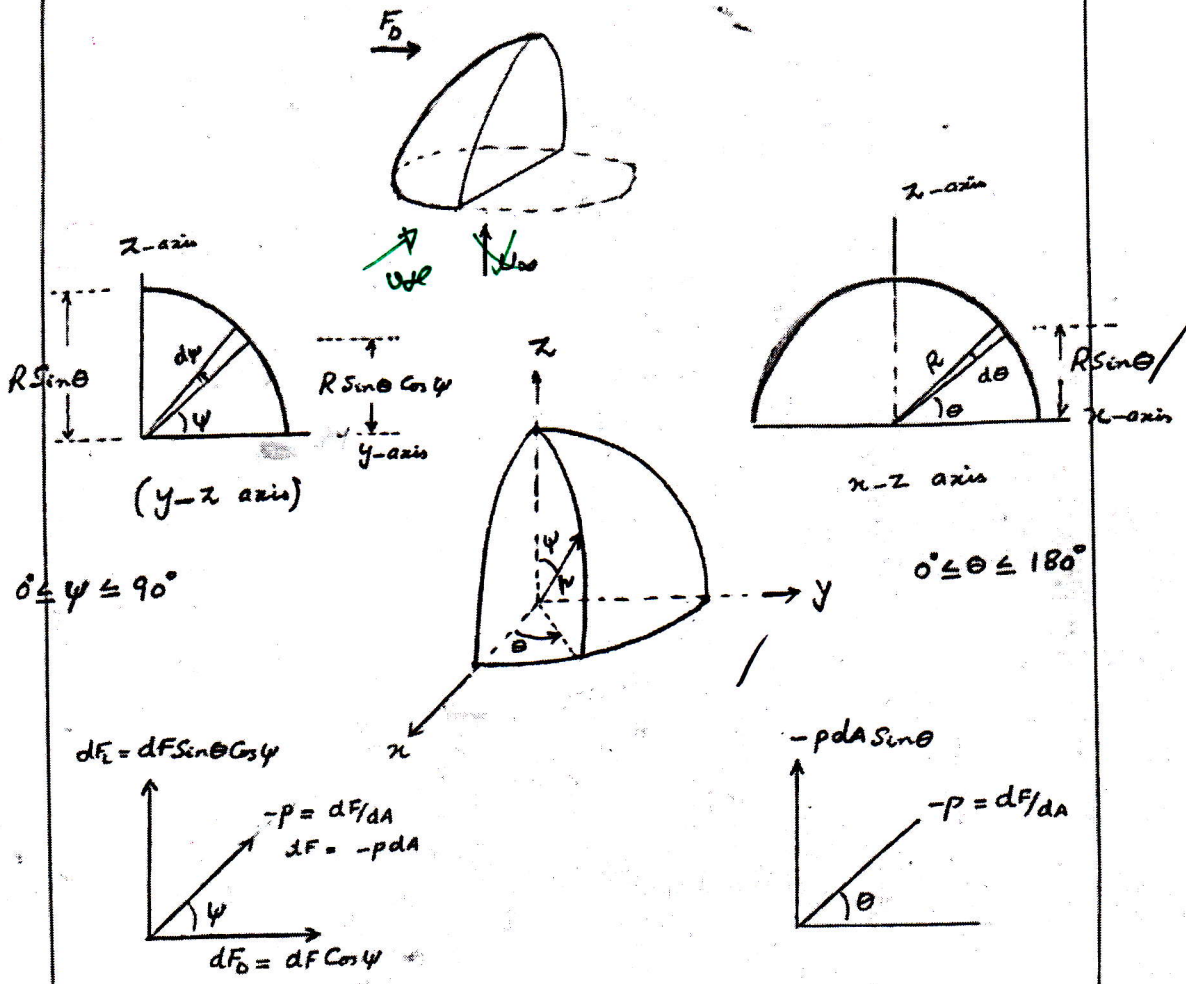
$$F_L = \frac{8}{3} \rho v_\infty^2 L R$$

$$F_L = \frac{8}{3} \left( \frac{1.2 \text{ kg}}{\text{m}^3} \right) \left( 100 \frac{\text{km}}{\text{hr}} * \frac{1000 \text{ m}}{1 \text{ km}} * \frac{1 \text{ hr}}{60^2 \text{ s}} \right)^2 4 \text{ m} * 1.63 \text{ m} = \boxed{16,200 \text{ N}}$$

$$16,200 \text{ N} * \left( 0.225 \frac{\text{lb}}{\text{N}} \right) = \boxed{3,650 \text{ lb}}$$

Problem # 2

Problem 4.4 p. 82 for quarter sphere. Integrate pressure and calculate the forces in all three directions.



$0 \leq \psi \leq 90^\circ$

$0 \leq \theta \leq 180^\circ$

$dF_x = dF \sin \theta \cos \psi$   
 $-p = dF/dA$   
 $dF = -p dA$   
 $dF_y = dF \cos \psi$

$-p dA \sin \theta$   
 $-p = dF/dA$

Find :  
 $F_z = ?$   
 $F_y = ?$   
 $F_x = ?$

3) Force in the z-direction  $F_z$ 

The pressure b/w  $z=0$  and any point on the quarter sphere can be obtained by applying Bernoulli's equation

$$\frac{P}{\gamma} + \frac{V^2}{2g} + z = \frac{P_0}{\gamma} + \frac{V_0^2}{2g} + 0$$

$$P = \gamma \left( \frac{P_0}{\gamma} + \frac{V_0^2}{2g} - z - \frac{V^2}{2g} \right)$$

$$\boxed{P = P_0 + \frac{\rho}{2} V_0^2 - \gamma z - \frac{\rho}{2} V^2} \quad \text{--- (A)}$$

This is the general equation for the pressure on the curved surface

We have

$$z = R \sin \theta \cos \psi$$

$$V = \frac{-3}{2} U_0 \sin \theta \quad \text{Velocity at any point of sphere}$$

$$P = P_0 + \frac{\rho}{2} U_0^2 - \gamma R \sin \theta \cos \psi - \frac{\rho}{2} \left( \frac{-3}{2} U_0 \sin \theta \right)^2$$

The force component on the curved surface in the z-direction can be estimated by integration as follows:

$$F_{z_s} = \int_F dF_z = \int_A -P dA \sin \theta \cos \psi dA$$

$$\text{Where } dA = R^2 \sin \theta d\psi d\theta$$

and the integral limits are  $\theta: 0$  to  $\pi$ , &  $\psi: 0$  to  $\pi/2$

$$F_{z_s} = - \int_0^\pi \int_0^{\pi/2} \left( P_0 + \frac{\rho}{2} U_0^2 - \gamma R \sin \theta \cos \psi - \frac{\rho}{2} \frac{9}{4} U_0^2 \sin^2 \theta \right) \sin \theta \cos \psi R^2 \sin \theta d\psi d\theta$$

$$F_{z_s} = - \int_0^\pi \int_0^{\pi/2} \left( P_0 + \frac{\rho}{2} U_0^2 - \gamma R \sin \theta \cos \psi - \frac{9}{8} \rho U_0^2 \sin^2 \theta \right) \sin^2 \theta \cos \psi R^2 d\psi d\theta$$

$$F_{z_s} = R^2 \left( -P_0 - \frac{\rho}{2} U_0^2 \right) \int_0^\pi \sin^2 \theta \left[ \int_0^{\pi/2} \cos \psi d\psi \right] d\theta$$

$$+ \gamma R^3 \int_0^\pi \sin^3 \theta \left[ \int_0^{\pi/2} \cos \psi d\psi \right] d\theta$$

$$+ \frac{9}{8} \rho U_0^2 R^2 \int_0^\pi \sin^4 \theta \left[ \int_0^{\pi/2} \cos \psi d\psi \right] d\theta$$

$$F_{z_s} = -\frac{1}{2} \rho_0 \pi R^2 - \frac{1}{4} \rho U_0^2 \pi R^2 + \frac{1}{3} \gamma \pi R^3 + \frac{9}{8} + \frac{3}{8} \rho U_0^2 \pi R^2$$

$$F_{z_s} = -\frac{1}{2} \rho_0 \pi R^2 - \frac{1}{4} \rho U_0^2 \pi R^2 + \frac{1}{3} \gamma \pi R^3 + \frac{27}{64} \rho U_0^2 \pi R^2 \quad (i)$$

Force on the base

$$\begin{aligned} F_{z_{base}} &= P_b \times \text{Area} & \text{Area} &= A_b = \frac{1}{2} \pi R^2 \\ &= \left( \rho_0 + \frac{\rho U_0^2}{2} - \frac{\rho V_b^2}{2} \right) \times \frac{\pi R^2}{2} \\ &= \frac{1}{2} \rho_0 \pi R^2 + \frac{1}{4} \rho U_0^2 \pi R^2 - \frac{1}{4} \rho V_b^2 \pi R^2 \end{aligned}$$

The net force in z-direction can be calculated by the addition of the component force on the base and the force on the curved surface

$$\begin{aligned} F_z &= F_{z_s} + F_{z_{base}} \\ &= -\frac{1}{2} \rho_0 \pi R^2 - \frac{1}{4} \rho U_0^2 \pi R^2 + \frac{1}{3} \gamma \pi R^3 + \frac{27}{64} \rho U_0^2 \pi R^2 \\ &\quad + \frac{1}{2} \rho_0 \pi R^2 + \frac{1}{4} \rho U_0^2 \pi R^2 - \frac{1}{4} \rho V_b^2 \pi R^2 \end{aligned}$$

$$F_z = \frac{1}{3} \gamma \pi R^3 + \frac{27}{64} \rho U_0^2 \pi R^2 - \frac{1}{4} \rho V_b^2 \pi R^2$$

## 2) Force in y-direction

The force on the curved surface in y-direction is computed by:

$$F_{y_s} = \int_F dF_y = \int_A -P dA \sin \theta \sin \psi dA$$

Where

$$dA = R^2 \sin \theta d\psi d\theta$$

and integral limits are  $\theta: 0$  to  $\pi$  &  $\psi: 0$  to  $\pi/2$

$$F_{y_s} = - \int_0^\pi \int_0^{\pi/2} \left( P_{\infty} + \frac{\rho U_{\infty}^2}{2} - \gamma R \sin \theta \cos \psi - \frac{\rho}{2} \left( -\frac{3}{2} U_{\infty} \sin \theta \right)^2 \right) \sin \theta \sin \psi R^2 \sin \theta d\psi d\theta$$

$$F_{y_s} = R^2 \left( -P_{\infty} - \frac{\rho U_{\infty}^2}{2} \right) \int_0^\pi \sin^2 \theta \left[ \int_0^{\pi/2} \sin \psi d\psi \right] d\theta \\ + \gamma R^3 \int_0^\pi \sin^3 \theta \left[ \int_0^{\pi/2} \sin \psi \cos \psi d\psi \right] d\theta \\ + \frac{9}{8} \rho U_{\infty}^2 R^2 \int_0^\pi \sin^4 \theta \left[ \int_0^{\pi/2} \sin \psi d\psi \right] d\theta$$

$$F_{y_s} = R^2 \left( P_{\infty} - \frac{\rho U_{\infty}^2}{2} \right) \int_0^\pi \sin^2 \theta [1] d\theta + \gamma R^3 \int_0^\pi \sin^3 \theta \left[ \frac{1}{2} \right] d\theta \\ + \frac{9}{8} \rho U_{\infty}^2 R^2 \int_0^\pi \sin^4 \theta [1] d\theta$$

$$F_{y_s} = R^2 \left( P_{\infty} - \frac{\rho U_{\infty}^2}{2} \right) \left( \frac{\pi}{2} \right) + \gamma R^3 \left( \frac{4}{3} \right) \left( \frac{1}{2} \right) + \frac{9}{8} \rho U_{\infty}^2 R^2 \left( \frac{3\pi}{8} \right) [1]$$

$$F_{y_s} = -\frac{1}{2} P_{\infty} \pi R^2 - \frac{1}{4} \rho U_{\infty}^2 \pi R^2 + \frac{2}{3} \gamma R^3 + \frac{9}{8} \times \frac{3}{8} \rho U_{\infty}^2 \pi R^2$$

$$F_{y_s} = -\frac{1}{2} P_{\infty} \pi R^2 - \frac{1}{4} \rho U_{\infty}^2 \pi R^2 + \frac{2}{3} \gamma R^3 + \frac{27}{64} \rho U_{\infty}^2 \pi R^2$$

Force on face of quarter sphere

$$F_{y_f} = \int dF_y = P A_y \quad (y = \text{Center of pressure})$$

$$F_{y_f} = \left( P_{\infty} + \frac{\rho}{2} \frac{U_{\infty}^2}{2} - \gamma z - \frac{\rho V_b^2}{2} \right) \left( \frac{\pi R^2}{2} \right)$$

$$F_{y_f} = P_{\infty} \frac{\pi R^2}{2} + \frac{\rho U_{\infty}^2}{2} \left( \frac{\pi R^2}{2} \right) - \gamma \left( \frac{4R}{3\pi} \right) \left( \frac{\pi R^2}{2} \right) - \frac{\rho V_b^2}{2} \left( \frac{\pi R^2}{2} \right)$$

$$F_{y_f} = P_{\infty} \frac{\pi R^2}{2} + \frac{\rho U_{\infty}^2 \pi R^2}{4} - \frac{2\pi R^3}{3} - \frac{\rho V_b^2 \pi R^2}{4}$$

The net force in y-direction can be calculated by summing the components forces on curved surface and forces on face of quarter circle

$$\begin{aligned} F_y &= F_{y_s} + F_{y_f} \\ &= \frac{-1}{2} P_{\infty} \pi R^2 - \frac{1}{4} \rho U_{\infty}^2 \pi R^2 + \frac{2}{3} \gamma R^3 + \frac{27}{64} \rho U_{\infty}^2 \pi R^2 \\ &\quad + \frac{P_{\infty} \pi R^2}{2} + \frac{\rho U_{\infty}^2 \pi R^2}{4} - \frac{2\pi R^3}{3} - \frac{\rho V_b^2 \pi R^2}{4} \end{aligned}$$

$$F_y = \frac{27}{64} \rho U_{\infty}^2 \pi R^2 - \frac{\rho V_b^2 \pi R^2}{4}$$

Further, if  $V_b \approx 0$  (assumed)

OK.

Velocity is 400  
inside is 400  
not at base

$$F_y = \frac{27}{64} \rho U_{\infty}^2 \pi R^2$$

OK

(3)

Force in  $x$ -direction

Force component on the curved surface in  $x$ -direction can be estimated as:

$$F_{x_s} = \int_F dF_x = \int_A -P dA \cos \theta dA$$

where

$$dA = R^2 \sin \theta d\psi d\theta$$

and integral limits are  $\theta: 0$  to  $\pi$ ,  $\psi: 0$  to  $\pi/2$

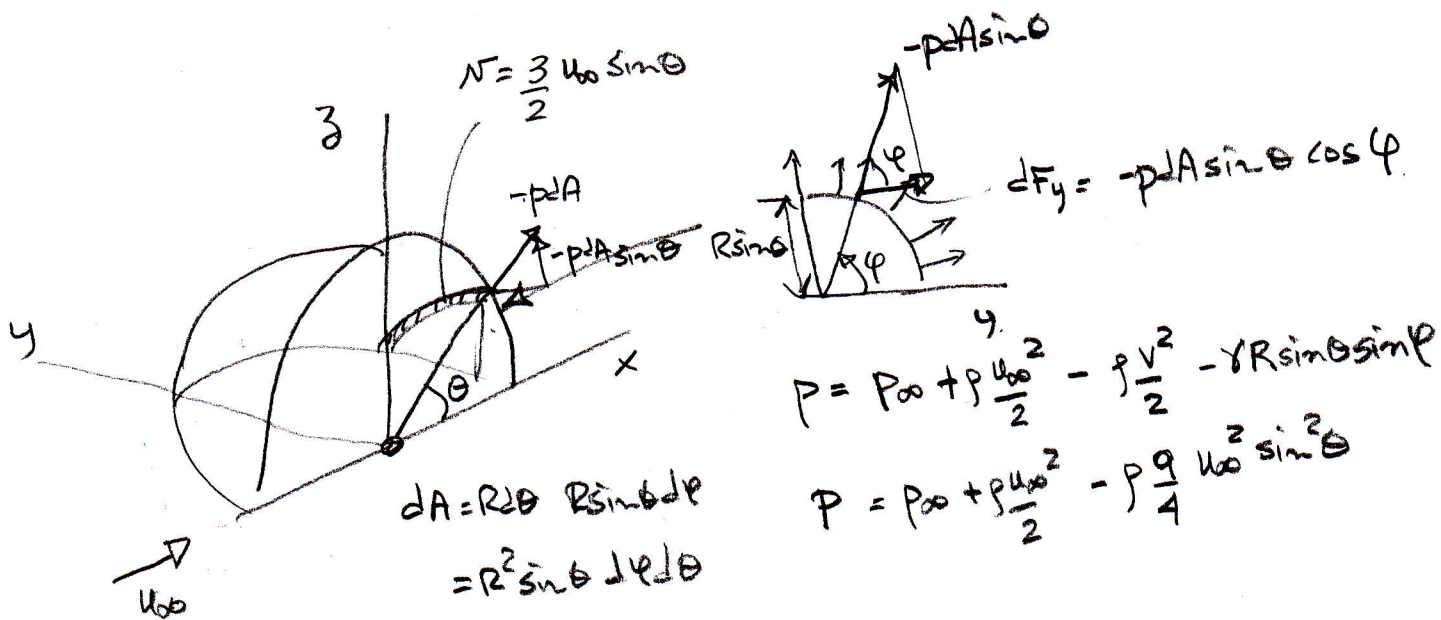
$$F_{x_s} = - \int_0^\pi \int_0^{\pi/2} \left( P_{\infty} + \frac{\rho}{2} U_{\infty}^2 - \gamma R \sin \theta \cos \psi - \frac{\rho}{2} \left( -\frac{3}{2} U_{\infty} \sin \theta \right)^2 \right) \cos \theta R^2 \sin \theta d\psi d\theta$$

$$\begin{aligned} F_{x_s} &= R^2 \left( -P_{\infty} - \frac{\rho}{2} U_{\infty}^2 \right) \int_0^\pi \cos \theta \sin \theta \left[ \int_0^{\pi/2} d\psi \right] d\theta \\ &\quad + \gamma R^3 \int_0^\pi \sin^2 \theta \cos \theta \left[ \int_0^{\pi/2} \cos \psi d\psi \right] d\theta \\ &\quad + \frac{9}{8} \rho U_{\infty}^2 R^2 \int_0^\pi \sin^3 \theta \cos \theta \left[ \int_0^{\pi/2} \cos^2 \psi d\psi \right] d\theta \end{aligned}$$

$$F_{x_s} = R^2 \left( -P_{\infty} - \frac{\rho}{2} U_{\infty}^2 \right) [0] + \gamma R^3 [0] + \frac{9}{8} \rho U_{\infty}^2 R^2 [0]$$

$$\boxed{F_{x_s} = 0}$$

Due to symmetrical geometry, the forces in  $x$ -direction will be zero.



Curved surface.

$$F_y = - \int_{\theta=0}^{\pi} \int_{\phi=0}^{\pi/2} \left( p_0 + \frac{\rho U_0^2}{2} - \frac{\rho \gamma U_0^2}{4} \sin^2 \theta + \gamma R \sin \theta \sin \phi \right) R^2 \sin \theta \, d\theta \, d\phi \sin \theta \cos \phi$$

$$\textcircled{1} = - p_0 R^2 \int_0^{\pi} \sin^2 \theta \left[ \int_0^{\pi/2} \cos \phi \, d\phi \right] d\theta = - p_0 \frac{\pi R^2}{2}$$

$$\textcircled{2} = - \frac{\rho U_0^2}{2} R^2 \int_0^{\pi} \sin^2 \theta \left[ \int_0^{\pi/2} \cos \phi \, d\phi \right] d\theta = - \frac{\rho U_0^2}{2} \frac{\pi R^2}{2}$$

$$\textcircled{3} = + \frac{\rho \gamma}{4} U_0^2 R^2 \int_0^{\pi} \sin^4 \theta \left[ \int_0^{\pi/2} \cos \phi \, d\phi \right] d\theta = \frac{27}{64} \rho \gamma U_0^2 \pi R^2$$

$$\textcircled{4} = + \gamma R^3 \int_0^{\pi} \sin^3 \theta \left[ \int_0^{\pi/2} \sin \phi \cos \phi \, d\phi \right] d\theta = \frac{2}{3} \gamma R^3$$



Fy.

On side.

$$v = u_{\infty}$$

$$P = P_{\infty} - \gamma n \sin \theta$$

$$F_{y \text{ side}} = \int_{R \pi} P dA$$

$$= \int_0^R \int_0^{\pi} (P_{\infty} - \gamma n \sin \theta) r d\theta dr$$

$$\textcircled{1} = P_{\infty} \int_0^R r \left[ \int_0^{\pi} d\theta \right] dr = P_{\infty} \frac{\pi R^2}{2}$$

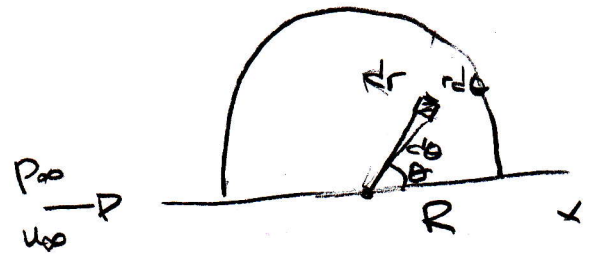
$$\textcircled{2} = -\gamma \int_0^R r \left[ \int_0^{\pi} \sin \theta d\theta \right] dr = -\gamma \frac{2R^3}{3} = -\frac{2}{3} \gamma R^3$$

Net Force in y

$$-\cancel{P_{\infty} \frac{\pi R^2}{2}} - \cancel{\frac{\rho u_{\infty}^2 \pi R^2}{4}} + \frac{27}{64} \rho u_{\infty}^2 \pi R^2 + \cancel{\frac{2}{3} \gamma R^3} + \cancel{P_{\infty} \frac{\pi R^2}{2}} - \cancel{\frac{2}{3} \gamma R^3}$$

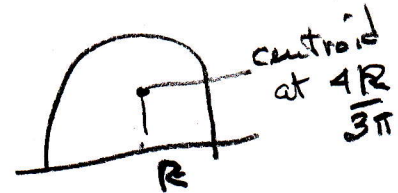
$$F_y = \frac{27}{64} \rho u_{\infty}^2 \pi R^2 - \frac{\rho u_{\infty}^2 \pi R^2}{4}$$

$$dA = r dr d\theta$$



Bernoulli

$$P_{\infty} + \frac{\rho u_{\infty}^2}{2} = P + \frac{\rho u^2}{2} - \gamma R \sin \theta$$



$$F = A \times P_{\text{centroid}}$$

$$\frac{\pi R^2}{2} \times \left( P_{\infty} + \frac{\gamma 4R}{3\pi} \right)$$

$$P_{\infty} \frac{\pi R^2}{2} - \frac{2}{3} \gamma R^3$$

Derive Rubey's fall velocity equation in equations (5.23 a and b) from combining equation (5.22 b) and  $C_D = 2 + 24/Re_p$

$$\omega_o = \left[ \frac{4}{3} (G-1) g \frac{d_s}{C_D} \right]^{1/2} \quad \text{--- 5.22 b}$$

$$C_D = 2 + \frac{24}{Re_p}$$

$$Re = \frac{\omega_o d_s}{\nu}$$

$$C_D = 2 + \frac{24\nu}{\omega_o d_s}$$

$$\omega_o^2 = \frac{4}{3} (G-1) g \frac{d_s}{C_D}$$

$$\omega_o^2 C_D = \frac{4}{3} (G-1) g d_s$$

$$\omega_o^2 \left( 2 + \frac{24\nu}{\omega_o d_s} \right) - \frac{4}{3} (G-1) g d_s = 0$$

$$2\omega_o^2 + \frac{24\nu}{d_s} \omega_o - \frac{4}{3} (G-1) g d_s = 0$$

$$ax^2 + bx + c = 0$$

$$a = 2 \quad b = \frac{24\nu}{d_s} \quad c = -\frac{4}{3} (G-1) g d_s$$

$$x = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

$$\omega_o = \frac{-\frac{24\nu}{d_s} \pm \sqrt{\left(\frac{24\nu}{d_s}\right)^2 - 4 \times 2 \left(-\frac{4}{3} (G-1) g d_s\right)}}{2 \times 2}$$

$$= \frac{-6\nu}{d_s} \pm \frac{1}{4} \sqrt{\frac{16 \times 36 \nu^2}{d_s^2} + 16 \times \frac{2}{3} (G-1) g \frac{d_s}{d_s^2} \times d_s^2}$$

$$= \frac{-6\nu}{d_s} \pm \frac{4}{4d_s} \sqrt{36\nu^2 + \frac{29}{3} (G-1) d_s^3}$$

$$\omega_0 = -\frac{6v}{ds} + \frac{1}{ds} \sqrt{\frac{2g}{3}(\sigma-1)ds^3 + 36v^2}$$

$$\omega_0 = \frac{1}{ds} \left[ \sqrt{\frac{2g}{3}(\sigma-1)ds^3 + 36v^2} - 6v \right] \quad \text{--- 5.23 a}$$

or

$$\omega_0 = -\frac{6v}{ds} + \frac{1}{ds} \sqrt{\frac{2g}{3}(\sigma-1)ds^3 + 36v^2}$$

$$= -\sqrt{\frac{36v^2}{ds^2}} \times \sqrt{\frac{(\sigma-1)gds}{(\sigma-1)gds}} + \sqrt{\frac{2g(\sigma-1)ds^3}{3ds^2} + \frac{36v^2}{ds^2} \frac{(\sigma-1)gds}{(\sigma-1)gds}}$$

$$= \sqrt{(\sigma-1)gds} \left[ \sqrt{\frac{2}{3} + \frac{36v^2}{(\sigma-1)gds^3}} - \sqrt{\frac{36v^2}{(\sigma-1)gds^3}} \right] \quad \text{--- 5.23 b}$$

$$\omega_0 = -\frac{6v}{ds} + \frac{1}{ds} \sqrt{\frac{2g}{3}(\sigma-1)ds^3 + 36v^2}$$

$$\omega_0 = \frac{1}{ds} \left[ \sqrt{\frac{2g}{3}(\sigma-1)ds^3 + 36v^2} - 6v \right] \quad \text{--- 5.23 a}$$

or

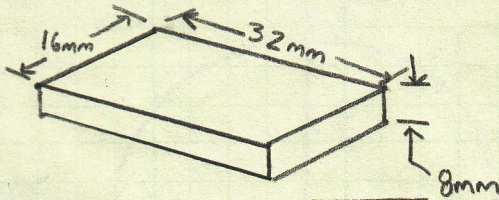
$$\omega_0 = -\frac{6v}{ds} + \frac{1}{ds} \sqrt{\frac{2g}{3}(\sigma-1)ds^3 + 36v^2}$$

$$= -\sqrt{\frac{36v^2}{ds^2}} \times \sqrt{\frac{(\sigma-1)gds}{(\sigma-1)gds}} + \sqrt{\frac{2g(\sigma-1)ds^3}{3ds^2} + \frac{36v^2}{ds^2} \frac{(\sigma-1)gds}{(\sigma-1)gds}}$$

$$= \sqrt{(\sigma-1)gds} \left[ \sqrt{\frac{2}{3} + \frac{36v^2}{(\sigma-1)gds^3}} - \sqrt{\frac{36v^2}{(\sigma-1)gds^3}} \right] \quad \text{--- 5.23 b}$$

PROBLEM 4

ESTIMATE THE VOLUME, SPHERICITY, & COREY SHAPE FACTOR OF THE PARTICLE.  
ESTIMATE THE SETTLING VELOCITY OF THE PARTICLE FROM FIGURE 5.2A & COMPARE  
WITH THE SETTLING VELOCITY OF AN EQUIVALENT VOLUME SPHERE.



$$V_s = 16(32)8 = 4096 \text{ mm}^3 = 4.10 \text{ cm}^3 = V_s$$

$$S_p = \left( \frac{l_c(l_b)}{l_a^2} \right)^{1/3} = \left( \frac{8(16)}{32^2} \right)^{1/3} = 0.5$$

$$d_s^3 = abc \Rightarrow d_s = \sqrt[3]{32(16)(8)}$$

FIGURE 4: Approximate Particle Diagram  
Assuming high  $Re_p$  gives:

$$C_o = \frac{l_c}{\sqrt{l_a l_b}} = \frac{8}{\sqrt{32(16)}} \Rightarrow C_o = 0.354$$

$$C_D = \frac{24}{Re_p} + \frac{0.5}{C_o^2} \Rightarrow C_D = \frac{0.5}{0.354^2} \Rightarrow C_D = 4$$

Correct

First iteration assuming high  $Re_p$  gives a fall velocity:

$$\omega = \left( \frac{4}{3}(G-1) \frac{g d_s}{\rho_0} \right)^{1/2} = \left( \frac{4}{3}(2.65-1) \frac{9.81(0.016)}{4} \right)^{1/2} \Rightarrow \omega = 0.293 \text{ m/s}$$

To verify the assumption of high  $Re_p$ , we can use the following:

$$Re_p = \frac{\omega d_s}{\nu} = \frac{0.293(0.016)}{1 \times 10^{-6}} \Rightarrow Re_p = 4701.06 > 10^3 \therefore Re_p \text{ Fully Turbulent}$$

Since our assumption of turbulent  $Re_p$ , the fall velocity is:

Correct

$$\omega_0 = 0.294 \text{ m/s}$$

For high  $Re_p$ , we can see that  $C_D = \frac{0.5}{C_o^2}$ , which plotted below on Figure 5:

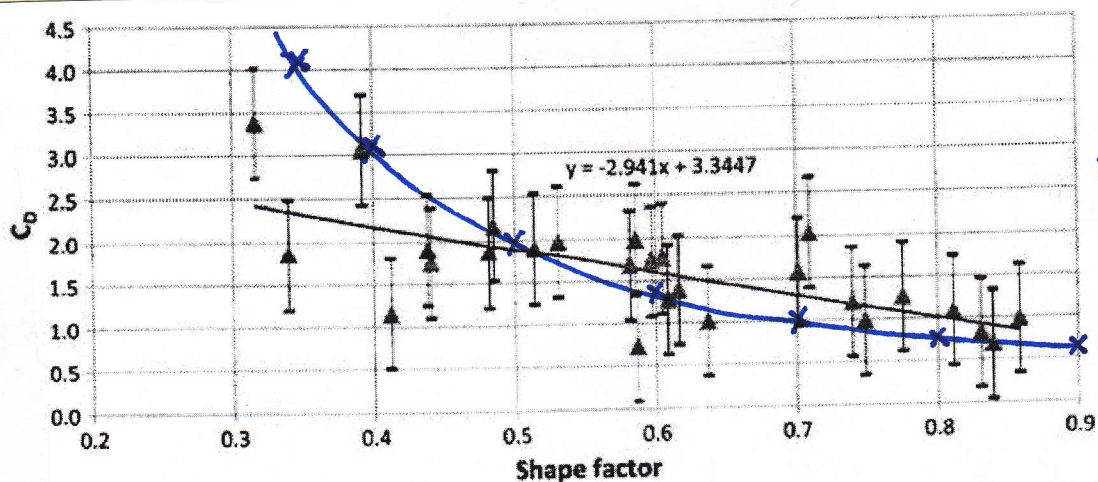


Fig. 5. Shape factor versus drag coefficient